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# Boiler Efficiency Analysis with Indirect Method PT. Indonesia Power UBP PLTU Lontar Unit 2

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#### **ABSTRACT**

Boiler efficiency is one of the important factors in determining the performance and environmental sustainability of PLTU. This study aims to analyze the factors that cause a decrease in boiler efficiency unit 2 PLTU Lontar. This study uses the indirect method (heat losses), it is found that the efficiency is 81.05%, and several large heat losses are found, namely L2: losses due to moisture content in coal of 5.01%, L3: losses due to hydrogen content in coal of 6.92% and comparison of heat losses data of PLTU Lontar boiler in 2015 and 2024. The analysis shows that the efficiency of the Lontar PLTU boiler has decreased from 82.96% in 2015 to 81.05% in 2024. This decrease is caused by several factors, namely: Increase in L3 (loss due to hydrogen content in coal) and L5 (loss due to unburned carbon). Decrease in L9 (loss due to sensible heat in fly ash). Decrease in coal quality. Efforts to improve boiler efficiency need to be made, such as, maintaining coal quality is very important to maintain boiler efficiency, and conducting regular boiler maintenance. The implementation of these measures is expected to improve the boiler efficiency of PLTU Lontar, reduce pollutant emissions, and improve environmental sustainability.

Keywords: Heat losses, boiler, efficiency, PLTU Lontar, coal

#### 1. INTRODUCTION

Boiler operating efficiency is of paramount importance in the power generation industry. This efficiency has a direct effect on productivity and environmental impact[1]. This research focuses on analyzing the boiler efficiency in Lontar Unit 2 PLTU owned by PT Indonesia Power UBP using indirect methods.

The indirect method, which is a common approach to boiler efficiency assessment, allows for quick and effective measurements using available data[2]. The significance of this study lies in providing insight into the factors affecting efficiency and the variables that can be optimized, thus enabling a comprehensive performance evaluation.

In addition, the study goes beyond simply calculating efficiency, but also assesses the viability of the boiler based on long-term operation. This holistic approach ensures sustainable and efficient boiler operation[3].

This research introduces a method of calculating heat losses based on ASME PTC 4.2013, a new approach to heat losses calculation. This method is in line with energy audit regulations, which emphasize efficiency and environmental impact [4].

The expected outcomes of this research include improved industry understanding of boiler efficiency using an indirect approach. comprehensive insights for PT Indonesia Power UBP PLTU Lontar Unit 2 in optimizing operations, practical recommendations for the company to improve boiler efficiency and performance, valuable scientific contributions to energy efficiency and PLTU operational efficiency, as well as a positive the operational efficiency and impact on sustainability of PT Indonesia Power UBP PLTU Lontar Unit 2 and the power generation industry as a whole.

Although this research focuses on boiler technical efficiency and optimization, rather than on economic analysis, it lays the foundation for further research on the economic aspects of boiler operation. The contribution of the research to improved technical performance and reduced environmental impact is significant, paving the way for a more sustainable and efficient power generation industry.

#### 2. METHODOLOGY

Data collection for this thesis was conducted using several methods, including:

#### (1) Literature Review

This method was employed to gather theories and formulas related to combustion performance in boilers. The results from this method were obtained from manuals, articles, journals, and other documents.

# (2) Numerical Data Processing

Primary data and data obtained from the literature review were processed numerically using formulas derived from the literature. Ten types of boiler losses were analyzed using the formula (1)[4].

$$h(\%) = 100 - (L_1 + L_2 + L_3 + L_4 + L_5 + L_6 + L_7 + L_8 + L_9 + L_{10})$$
(1)

where:

L1 : Dry flue gas loss (%)

L2 : Moisture loss in fuel (%)

L3 : Hydrogen content loss in fuel (%)

L4 : Loss due to moisture content in supply

air (%)

L5 : Loss due to unburned carbon (%)

L6 : Radiation and convection losses of the

boiler (value predetermined)

L7 : Unmeasured loss (value predetermined

by the manufacturer)

L8 : Loss due to sensible heat in bottom ash

(%)

L9 : Loss due to sensible heat in fly ash (%)

L10: Loss due to carbon monoxide formation

from incomplete combustion (%)

# (3) Consultation

Consultations were conducted to obtain more indepth information related to the research. These consultations were carried out through discussions or interviews with operators, employees, supervisors, and experts in the field.

The data analysis method used in this research involved collecting flue gas test data from the Air Pre-Heater (APH), selecting relevant data for boiler efficiency calculations in unit Y, performing

calculations using the specified ASME PTC 4-2013, and carefully calculating to minimize errors[4]. The final stage of data processing involved analyzing the obtained results to confirm their accuracy, then comparing them with related journals that conducted research on similar boilers to identify factors related to the research object.

#### **Direct Method**

Efficiency serves as a benchmark for a machine's performance. For a steam boiler engine, efficiency is defined as the comparison between the output energy (the steam produced) and the input energy (the fuel consumed). This is often called the direct or input-output method, as the efficiency value is calculated by dividing the energy output by the heat input[5].

With:

mMS : mass main steam (t/h)

hMS : enthalpy of main steam (kCal/kg) mFW : feedwater mass flow rate (t/h) hFW : feedwater enthalpy (kCal/kg)

mSHS : mass flow rate of spray superheater

(t/h)

hSHS: enthalpy of spray superheater

(kCal/kg)

mf : mass flow rate of fuel (t/h)

HHV : High Heating Value of fuel (kCal/kg)

The direct method is a suitable choice for quick and easy boiler efficiency calculations. Its advantages lie in the fewer parameters required and the ease of measurement. However, the weakness of this method is its inability to detail each heat loss occurring in the boiler. In addition, the accuracy of the mass flow rate and fuel calorific value measuring instruments significantly affects the accuracy of boiler efficiency calculations. For example, if the actual boiler efficiency is 90% but there is a 1% measurement error, the obtained efficiency value will be 90% ± 0.9%, which is between 89.1% and 90.9%.

#### **Indirect Method**

The indirect method provides a more in-depth understanding of the overall boiler performance. By knowing the value of each heat loss, operators can understand how factors such as fuel quality, operating conditions, and boiler design affect efficiency[6].

The process of calculating efficiency using the indirect method is as follows:

1. The first step is to calculate various heat losses occurring in the boiler, such as heat loss with flue gas, heat loss due to radiation, and heat loss due to leaks.

- 2. All values of individual heat losses are then summed up to obtain the total heat loss of the entire boiler.
- 3. Finally, boiler efficiency is obtained by subtracting the total heat loss from 100%.

The direct and indirect methods are two main approaches to calculating boiler efficiency. Each method has its own advantages and disadvantages. The direct method offers simplicity and ease of calculation. This method only requires a few parameters and can be done quickly. However, this method has limitations in terms of detail and accuracy. This method cannot determine the magnitude of each heat loss occurring in the boiler, and its accuracy depends on the accuracy of the measuring instruments[7].

On the other hand, the indirect method provides more complete information about boiler performance. This method allows for knowing the material and energy balance in each part of the boiler, thus helping to identify areas that can be improved to increase efficiency[8]. However, this method requires a long time and laboratory facilities for the analysis of fuel and flue gas samples.

The weakness of the direct method can be overcome by the indirect method, which calculates various heat losses in the boiler [9]. Efficiency can be obtained by subtracting the total heat loss from 100%. A significant advantage of this method is the minimal influence of measurement errors on efficiency calculations[10]. This is because the calculated heat loss is a small part of the entire boiler system. For example, if the boiler heat loss value is 10% and the indirect method has an error of 1%, then the actual boiler heat loss becomes 10%  $\pm$  0.1% = 9.9% to 10.1%. This means that the boiler efficiency is between 89.9% and 90.1%.

The heat loss method is also known as the indirect method. ASME PTC-4 has issued a standard for calculating boiler efficiency using the heat loss method, with the latest revision in 2013[4]. The calculation of heat losses is shown by equations (2)-(7).

# Loss due to dry flue gas (L<sub>1</sub>)

$$L_1 = \frac{HDFgLvCr \ x \ MFrDFg}{HHV} \ x \ 100\% \tag{2}$$

Where:

HDFgLvCr : enthalpy of dry flue gas leaving

the air preheater (excluding air leakage in the air preheater)

(kJ/kg)

MFrDFg : mass of dry flue gas leaving the

boiler (kg/kg-fuel)

HHV : High Heating Value of coal (kJ/kg-

fuel)

## Loss due to moisture content in coal (L<sub>2</sub>)

$$L_2 = \frac{MFrWF \ x \ (HstLvCr - Hw)}{HHV} \ x \ 100\% \tag{3}$$

Where:

MFrWF : moisture content in fuel (kg/kg-

fuel)

HstLvCr : enthalpy of steam (evaporation of

water) leaving the air preheater (excluding air leakage in the air

preheater) at 1 psia (kJ/kg)

HW : enthalpy of water at reference air

temperature of 33°C (kJ/kg)

# Loss due hidrogen content in coal (L<sub>3</sub>)

$$L_3(\%) = \frac{MfrWH2F \ x \ (HstLvCr - Hw)}{HHV} \ x \ 100 \ \ (4)$$

Where:

MFrWF : moisture content in fuel (kg/kg-

fuel)

HstLvCr : enthalpy of steam (evaporation of

water) leaving the air preheater (excluding air leakage in the air preheater) at 1 psia (kJ/kg)

HW : enthalpy of water at reference air

temperature of 33°C (kJ/kg)

# Loss due moinsture content in air supply (L4)

$$L_4(\%) = \frac{MfrWH2F \ x \ (HstLvCr - Hw)}{HHV} \ x \ 100 \ \ (5)$$

Where:

MFrWA : moisture content in air supply

(kg/kg-fuel)

HWvLvCr : enthalpy of steam leaving the air

preheater with no air leakage

(kJ/kg)

# Loss due unburn carbon content in coal (L<sub>5</sub>)

$$L_5(\%) = MpUbC x \frac{HHVCRs}{HHV} x 100$$
 (6)

Where:

MpUbC : unburn content in coal, % mass HHVCRs : heating value of carbon residue

33.700 kJ/kg

**Losses due to radiation and convection** from the boiler wall surface, the value of which is specified by the boiler manufacturer (%).

**Unaccounted losses**, the value of which is specified by the boiler manufacturer (%).

#### Losses due to sensible heat in bottom ash (L<sub>8</sub>)

$$L_8(\%) = \frac{xUcb \ x \ MFrR \ x \ Hcba}{HHV} \ x \ 100 \tag{7}$$

Where:

xUcb : losses due to sensible heat in bottom

ash

MFrR : mass of ash residue from combustion

(kg/kg-fuel)

Hcba : entalpi bottom ash (kJ/kg)

## Losses due to sensible heat in fly ash (L9)

$$L_9(\%) = \frac{xUcf \ x \ MFrR \ x \ Hcfa}{HHV} \ x \ 100 \tag{8}$$

Where:

xUcf : ratio of fly ash to total ash

MFrR : mass of residual ash from combustion

(kg/kg-fuel)

Hcfa : enthalpy of fly ash (kJ/kg)

# Losses due to carbon monoxide formation from incomplete combustion ( $L_{10}$ )

$$L_{10}(\%) = \frac{23630 \text{ kJ } x \text{ DVpCO } x \frac{Mpcb}{(DVpco + DVpco_2)}}{HHV} x 100 (9)$$

Where:

MpCb : carbon burned (%)

DVpCO : CO concentration in flue gas at air

preheater outlet (%)

 $\mathsf{DVpCO}_2$ :  $\mathsf{CO}_2$  concentration in flue gas at air

preheater outlet (%)

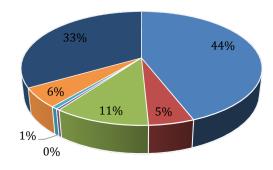
# 3. RESULTS AND DISCUSSION

Boiler performance data is crucial in evaluating the efficiency and operational performance of a boiler in a power plant. Analysis of this data provides a clear picture of how the boiler operates, including combustion efficiency, fuel consumption, and overall system performance. By understanding boiler performance data, companies can identify areas for improvement to enhance operational efficiency and reduce environmental impact.

This sub-chapter provides a comprehensive overview of boiler performance data at Boiler Unit 2 of PT. Banten 3 Lontar POMU and an in-depth analysis of its implications for overall power plant operations. Boiler performance data for February 2024 at 100% load.

**Table 1**. Numerical calculation results using the transfer matrix method

No	Boiler performance	Unit	Value
1	CO <sub>2</sub> Content	%	12.87
2	O <sub>2</sub> Content	%	3.64
3	Flue Gas temprature	°C	160.19
4	Ambient Temperature	°C	30
5	Air Moisture	kg/kg	0.02
6	Radiation Loses	%	2.5
7	Fuel Higher heating Value	kcal/kg	4,223.51



- Carbon Content (AR)
- Hydrogen Content (AR)
- Oxygen Content (AR)
- Sulfur Content (AR)
- Nitrogen Content (AR)
- Ash Content (AR)
- Moisture Content (AR)

Figure 1. Chart content

Primary and secondary data from the literature review were numerically processed using the formulas derived from the literature. This study only analyzed 10 types of losses in boilers. The calculation of boiler efficiency with heat loss used the following equation.

$$\eta(\%) = 100 - (L_1 + L_2 + L_3 + L_4 + L_5 + L_6 + L_7 + L_8 + L_9 + L_{10})$$

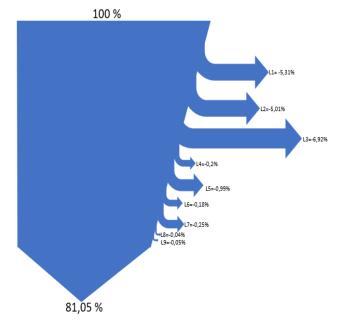


Figure 2. Loses diagram

Analysis of Boiler Efficiency Calculation Results, Unit 2, PT Indonesia Power UBP Banten 3 Lontar. Based on the provided data, the following is an analysis of the calculated efficiency of Boiler Unit 2, PT Indonesia Power UBP Banten 3 Lontar:

# **Heat Losses**

- Dry gas loss (L1): 5.31%
- Moisture in coal loss (L2): 5.01%
- Hydrogen in coal loss (L3): 6.92%

- Moisture in combustion air loss (L4): 0.20%
- Unburned carbon loss (L5): 0.99%
- Radiation and convection losses from boiler wall surface (L6): 0.18% (manufacturer's specified)
- Unaccounted losses (L7): 0.25% (manufacturer's specified)
- Sensible heat loss of bottom ash (L8): 0.04%
- Sensible heat loss of fly ash (L9): 0.05%
- CO loss (L10): 0.00%

#### **Total Losses**

Total heat loss in Boiler Unit 2, PT Indonesia Power UBP Banten 3 Lontar is:  $L_1$  +  $L_2$  +  $L_3$  +  $L_4$  +  $L_5$  +  $L_6$  +  $L_7$  +  $L_8$  +  $L_9$  +  $L_{10}$  = 18.95%

### **Boiler Efficiency**

Boiler efficiency can be calculated using the formula: Efficiency (%) = 100% - Total Losses (%). Therefore, the efficiency of Boiler Unit 2, PT Indonesia Power UBP Banten 3 Lontar is (100%-18.95%) = 81.05%

Based on the analysis above, it can be concluded that the efficiency of Boiler Unit 2, PT Indonesia Power UBP Banten 3 Lontar is 81.05%. This value is considered good and indicates that the boiler is operating quite efficiently. However, it should be noted that there is still 18.95% of energy lost as heat. This suggests that there is still potential to improve boiler efficiency by optimizing several aspects, such as:

- Reducing dry gas loss (L1): This can be done by improving coal quality or using more advanced combustion technology.
- Reducing moisture in coal loss (L2): This can be done by drying the coal before use.
- Reducing hydrogen in coal loss (L3): This can be done by selecting coal types with lower hydrogen content.
- Reducing moisture in combustion air loss (L4): This can be done by using a dehumidifier to reduce moisture in the combustion air.
- Reducing unburned carbon loss (L5): This can be done by improving combustion quality or using more advanced combustion technology.

By optimizing these aspects, it is expected that the efficiency of Boiler Unit 2, PT Indonesia Power UBP Banten 3 Lontar can be improved, thus saving energy and operating costs.

A comparative analysis of boiler efficiency at Unit 2 of PT. Indonesia Power UBP PLTU Lontar is presented, encompassing the initial design, a 2015 journal study on Unit 3, and a recent evaluation in February 2022. The primary goal is to discern efficiency trends over time and pinpoint the underlying causes. Insights from this study will serve as a roadmap for optimizing boiler performance and efficiency in future operations.

Detailed efficiency data and influencing parameters will be provided.

**Table 2**. Numerical calculation results using the transfer matrix method

Boiler model   DG1025/17.4-II13	matrix metriod				
Parameter	Boiler model DG1025/17.4-II13				
Superheated steam         Max. Continuous Evaporation (t/h)         1025         976.2           Outlet Pressure (MPa)         17.4         17.32           Outlet Temperature (°C)         541         541           Reheated steam         Flow (t/h)         839.4         802           Inlet/Outlet Pressure (MPa)         3.76/3.58         3.59/3.41           Inlet/Outlet Temperature (°C)         329/541         324/541           Flue Gas Temperature of Corrected (°C)         131         131           Feedwater Temperature (°C)         281         278           Drum Pressure (MPa)         18.77         18.77	Manufactur	Dong Fang Boiler Group Co.Ltd			
Superheated steam		Parameter	BMCR	BRL	
Superheated steam         Max. Continuous Evaporation (t/h)         1025         976.2           Outlet Pressure (MPa)         17.4         17.32           Outlet Temperature (°C)         541         541           Reheated steam         Flow (t/h)         839.4         802           Inlet/Outlet Pressure (MPa)         3.76/3.58         3.59/3.41           Inlet/Outlet Temperature (°C)         329/541         324/541           Temperature of Corrected (°C)         541         541           Flue Gas         131         131           Temperature of Corrected (°C)         541         541           Feedwater Temperature (°C)         281         278           Drum Pressure (MPa)         18.77         18.77           MPa)         18.77         18.77			1 0	1	
Evaporation (t/h)					
(t/h)         Outlet Pressure (MPa)         Outlet Pressure (°C)         Reheated steam         Flow (t/h)       839.4       802         Inlet/Outlet       3.76/3.58       3.59/3.41         Pressure (MPa)         Inlet/Outlet       329/541       324/541         Temperature (°C)         Flue Gas       131       131         Temperature of Corrected (°C)         Feedwater       281       278         Temperature (°C)         Drum Pressure (MPa)		Max. Continuous	1025	976.2	
Outlet Pressure (MPa) Outlet 541 541 Temperature (°C) Reheated steam Flow (t/h) 839.4 802 Inlet/Outlet 3.76/3.58 3.59/3.41 Pressure (MPa) Inlet/Outlet 329/541 324/541 Temperature (°C) Flue Gas 131 131 Temperature of Corrected (°C) Feedwater 281 278 Temperature (°C) Drum Pressure 18.77 18.77 (MPa)	steam				
(MPa)       Outlet     541     541       Temperature       (°C)       Reheated steam     Flow (t/h)     839.4     802       Inlet/Outlet     3.76/3.58     3.59/3.41       Pressure (MPa)       Inlet/Outlet     329/541     324/541       Temperature       (°C)     Flue Gas     131     131       Temperature of Corrected (°C)       Feedwater     281     278       Temperature       (°C)     Drum Pressure     18.77     18.77       (MPa)     18.77		(t/h)			
Outlet Temperature (°C)  Reheated steam  Flow (t/h) 839.4 802  Inlet/Outlet 3.76/3.58 3.59/3.41  Pressure (MPa)  Inlet/Outlet 329/541 324/541  Temperature (°C)  Flue Gas 131 131  Temperature of Corrected (°C)  Feedwater 281 278  Temperature (°C)  Drum Pressure 18.77 18.77  (MPa)		Outlet Pressure	17.4	17.32	
Temperature (°C)  Reheated steam  Flow (t/h) 839.4 802  Inlet/Outlet 3.76/3.58 3.59/3.41  Pressure (MPa)  Inlet/Outlet 329/541 324/541  Temperature (°C)  Flue Gas 131 131  Temperature of Corrected (°C)  Feedwater 281 278  Temperature (°C)  Drum Pressure 18.77 18.77  (MPa)		(MPa)			
(°C)         Reheated steam       Flow (t/h)       839.4       802         Inlet/Outlet Pressure (MPa)         Inlet/Outlet Temperature (°C)       329/541       324/541         Flue Gas       131       131         Temperature of Corrected (°C)       281       278         Temperature (°C)       Drum Pressure (MPa)       18.77       18.77		Outlet	541	541	
Reheated steam         Flow (t/h)         839.4         802           Inlet/Outlet Pressure (MPa)         3.76/3.58         3.59/3.41           Inlet/Outlet Temperature (°C)         329/541         324/541           Flue Gas         131         131           Temperature of Corrected (°C)         281         278           Temperature (°C)         Drum Pressure (MPa)         18.77         18.77		Temperature			
Inlet/Outlet		( -)			
Pressure (MPa)         Inlet/Outlet       329/541       324/541         Temperature       (°C)       131       131         Flue Gas       131       131       131         Temperature of Corrected (°C)       281       278         Temperature       (°C)       281       278         Temperature       (°C)       18.77       18.77         (MPa)       18.77       18.77	Reheated		839.4		
Inlet/Outlet 329/541 324/541 Temperature (°C) Flue Gas 131 131 Temperature of Corrected (°C) Feedwater 281 278 Temperature (°C) Drum Pressure 18.77 18.77 (MPa)	steam	Inlet/Outlet	3.76/3.58	3.59/3.41	
Temperature (°C)  Flue Gas 131 131  Temperature of Corrected (°C)  Feedwater 281 278  Temperature (°C)  Drum Pressure 18.77 18.77 (MPa)		Pressure (MPa)			
(°C)         Flue Gas       131       131         Temperature of       Corrected (°C)       281       278         Temperature       (°C)       Drum Pressure       18.77       18.77         (MPa)       18.77       18.77       18.77		Inlet/Outlet	329/541	324/541	
Flue Gas 131 131 Temperature of Corrected (°C) Feedwater 281 278 Temperature (°C) Drum Pressure 18.77 18.77 (MPa)		•			
Temperature of Corrected (°C)  Feedwater 281 278  Temperature (°C)  Drum Pressure 18.77 18.77 (MPa)		(°C)			
Corrected (°C) Feedwater 281 278 Temperature (°C) Drum Pressure 18.77 18.77 (MPa)		Flue Gas	131	131	
Feedwater 281 278 Temperature (°C) Drum Pressure 18.77 18.77 (MPa)					
Temperature (°C) Drum Pressure 18.77 18.77 (MPa)					
(°C) Drum Pressure 18.77 18.77 (MPa)		Feedwater	281	278	
Drum Pressure 18.77 18.77 (MPa)					
(MPa)					
		21411111000410	18.77	18.77	
Design Efficiency 93.26 93.71					
,		Design Efficiency	93.26	93.71	
(%)		(%)			

The Figure 3 illustrates the trend of boiler efficiency at PLTU Lontar from its initial operation in 2011 (93.26%) to 2015 (82.96%) and 2024 (81.05%). The analysis reveals a significant disparity between the design efficiency and the actual performance in subsequent years. To understand the factors contributing to the decline in efficiency between 2015 and 2024 and to devise strategies for maintaining optimal efficiency, a heat loss analysis will be conducted.

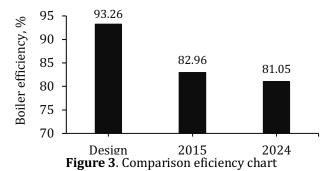


Table 3. Comparison boiler heat losses

Parameter	Unit	2015	2024
L1: Flue gas loss	%	6.13	5.31
L2: Moisture loss in coal	%	4.92	5.01
L3: Hydrogen loss in coal	%	4.98	6.92

-			
Parameter	Unit	2015	2024
L4: Moisture loss in	%	0.32	0.2
combustion air			
L5: Carbon in refuse loss	%	0	0.99
L6: Radiation and convection	%	0.18	0.18
losses (manufacturer's			
specified)			
L7: Unaccounted losses	%	0.25	0.25
(manufacturer's			
specified)			
L8: Sensible heat loss in	%	0.04	0.04
bottom ash			
L9: Sensible heat loss in fly	%	0.55	0.05
ash			
L10: Carbon monoxide loss	%	0	0
Boiler Efficiency	%	82.67	81.05

Based on the data presented in the Table 3, there were several changes in boiler heat losses between 2015 and 2024. The following is a comparative analysis:

#### **Decrease in Heat Losses**

- L1 (losses due to dry gas): decreased from 6.13 to 5.31%. This indicates more complete combustion and better heat transfer efficiency in 2024.
- L4 (losses due to moisture content in combustion air): decreased from 0.32 to 0.20%. This indicates drier combustion air and more efficient combustion in 2024.
- L6 (losses due to radiation and convection from the boiler wall surface): A constant value of 0.18% indicates that this factor does not affect the overall change in heat losses.
- L7 (unmeasured losses): A constant value of 0.25% indicates that this factor does not affect overall changes in heat losses.
- L8 (losses due to sensible heat in bottom ash):
   A constant value of 0.04% indicates that this factor does not affect overall changes in heat losses.
- L9 (losses due to sensible heat in fly ash): Significantly decreased from 0.55% to 0.05%. This indicates better fly ash processing and higher combustion efficiency in 2024.
- L10 (losses due to carbon monoxide formation caused by incomplete combustion): A constant value of 0.00% indicates that complete combustion was achieved in both years.

# **Increased Heat Losses**

• L2 (losses due to moisture content in coal): increased slightly from 4.92% to 5.01%. This may be due to variations in the moisture content of the coal used in 2024.

- L3 (losses due to hydrogen content in coal): increased significantly from 4.98% to 6.92%. This may be due to changes in the composition of coal used in 2024.
- L5 (losses due to unburned carbon): increased significantly from 0.0% to 0.99%. This indicates incomplete combustion in 2024, which could be caused by several factors such as suboptimal burner settings, low coal quality, or suboptimal boiler operating conditions.

Based on the increase in heat losses above, the researchers attempted to compare the coal composition used at the two different times. After comparing the coal composition, it was proven that the decrease in efficiency in 2024 was largely due to the decline in coal quality, as indicated by the coal composition used in 2024. In 2024, the water and hydrogen content in the coal was higher compared to the coal composition in 2015. Additionally, the unburned carbon loss indicates low coal quality, as shown in the Table 4.

**Table 4**. Compare coal content in boiler

No	Coal content	Unit	2015	2024
1	Carbon Content	%	46.20	44.19
	(AR)			
2	Hydrogen	%	3.59	5.16
_	Content (AR)			
3	Oxygen Content	%	12.64	10.85
	(AR)	0.4	0.46	0.40
4	Sulfur Content	%	0.46	0.42
5	(AR)	0/	0.70	0.69
5	Nitrogen Content (AR)	%	0.70	0.69
6	Ash Content	%	3.71	5.56
U	(AR)	70	5.71	5.50
7	Moisture	%	32.70	33.38
•	Content (AR)	, 0	0	22.00
8	HHV of Coal	kCal/kg	4,547	4,223.51

#### 4. CONCLUSION

A comparative analysis of the boiler efficiency of Unit 2 at PT. Indonesia Power Banten 3 Lontar revealed a decline from the initial design efficiency to the efficiency achieved in 2024. A further comparison with a similar boiler in 2015 showed a similar trend. To investigate the root causes of this efficiency degradation, a detailed analysis of coal quality was performed. The results indicated that the deterioration in coal quality, specifically the higher moisture and hydrogen content in the coal used in 2024, was a major contributing factor.

To mitigate this issue and improve boiler efficiency, several strategies can be adopted, including:

- 1) Utilizing higher quality coal with lower moisture and ash content and higher carbon content.
- 2) Implementing a rigorous maintenance program for the boiler system, focusing on the heating, combustion, and heat removal systems.
- 3) Adopting advanced technologies and more efficient fuels to enhance boiler performance.

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#### REFERENCES

- Camaraza-Medina Y, Retirado-Mediaceja Y, Hernandez-Guerrero A, Luis Luviano-Ortiz J. Energy efficiency indicators of the steam boiler in a power plant of Cuba. Thermal Science and Engineering Progress. 2021; 23(100880).
  - http://dx.doi.org/10.1016/j.tsep.2021.100880.
- Shahab OA, Amna S. Efficiency Analysis of Fire Tube Boiler Type at Refinery Utility Unit Center for Oil and Gas Human Resources Development (PPSDM Migas) Cepu. 2023; 2(7): 3109-3118.
  - http://dx.doi.org/10.53625/jcijurnalcakrawalailmiah.v2i7.5401.
- Ainun F, Jamaaluddin J. Analisa Efisiensi Economizer Terhadap Boiler (Gas Dan Solar) di PT. Spindo III, Tbk. Journal of Electrical and Electronic Engineering. 2018; 2(2): 99–104. https://doi.org/10.21070/jeee-u.v2i2.1697.
- 4. Fired Steam Generators: Performance Test Codes. Available: <a href="https://www.spic.ir">www.spic.ir</a>
- 5. Mulyani Y, Dhamayanthie I, Hilmi MI. The Efficiency of Water Tube Boiler Performance Using Direct Method on the Utility Unit. Riwayat: Educational Journal of History and Humanities. 2023; 6(4): 3232–3238. https://doi.org/10.24815/jr.v6i4.36824.
- Febriani SDA, Purwanto MR. Analysis of Boiler Engine Efficiency Unit 2 PT. PJB UP Paiton. Journal of Physics: Conference Series, IOP Publishing. 2021; 1805(1):012015. http://dx.doi.org/10.1088/1742-6596/1805/1/012015.
- Mojica-Cabeza CD, García-Sánchez CE, Silva-Rodríguez R, García-Sánchez L. A review of the different boiler efficiency calculation and modeling methodologies. Informador Técnico. 2021; 86(1).
  - http://dx.doi.org/10.23850/22565035.3697

- Rasworo P. Analisis efisiensi boiler CFB 2x60 MW PLTU PT X unit 2 dengan metode direct dan indirect. Journal of New Energies & Manufacturing. 2022; 1(2). <a href="https://dx.doi.org/10.22441/jonem.v1i2.14791">https://dx.doi.org/10.22441/jonem.v1i2.14791</a>.
- Zaman MR, Sinaga N. Thermal Losses Evaluation In 660 MW Coal-Fired Power Plant Using Indirect Efficiency Method. Jurnal Sains dan Teknologi Reaksi. 2021; 19(01): 1-9. http://dx.doi.org/10.30811/jstr.v19i01.2263.
- Lahijani AM, Supeni E, Kalantari F. A Review of Indirect Method for Measuring Thermal Efficiency in Fire Tube Steam Boilers. Journal of Industrial Pollution Control. 2018; 34(1): 18-32
- 11. Hendri, Suhengki, Ramdhan P. Analisa Efisiensi Boiler dengan Metode Heat Loss Sebelum dan Sesudah Overhaul PT. Indonesia Power UBP PLTU Lontar Unit 3. Jurnal Power Plant; 2017; 4(4): 218-227.